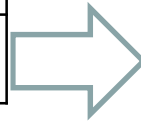


ME265: Thermal Engineering & Heat Transfer

Chapters
1. Energy Scenario
2. Thermodynamics
3. Mechanical Devices & Systems
4. Heat Transfer



4.1 Introduction	
4.2 Conduction	
4.3 Convection	4.3.1 Convection Fundamentals 4.3.2 External Forced Convection 4.3.3 Internal Forced Convection 4.3.4 Natural Convection
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4.3.2 External Forced Convection

□ For Parallel flow over flat plates

Flow over an isothermal flat plate

Parameter	Laminar flow $Re_x < 5 \times 10^5$	Turbulent Flow $5 \times 10^5 \leq Re_x \leq 10^7$
Local Friction Coefficient, $C_{f,x}$	$C_{f,x} = \frac{0.664}{Re_x^{1/2}}$	$C_{f,x} = \frac{0.059}{Re_x^{1/5}}$
Local Convection Coefficient, Nu_x	$Nu_x = 0.332 Re_x^{1/2} Pr^{1/3}$ For $Pr > 0.6$	$Nu_x = 0.0296 Re_x^{0.8} Pr^{1/3}$ For $0.6 \leq Pr \leq 60$

Parameter	Laminar flow $Re_x < 5 \times 10^5$	Turbulent Flow $5 \times 10^5 \leq Re_x \leq 10^7$
Average Friction Coefficient, C_f	$C_f = \frac{1.33}{Re_L^{1/2}}$	$C_f = \frac{0.074}{Re_L^{1/5}}$
Avg. Convection Coefficient, Nu	$Nu = 0.664 Re_L^{1/2} Pr^{1/3}$ For $Pr > 0.6$	$Nu = 0.037 Re_L^{0.8} Pr^{1/3}$ For $0.6 \leq Pr \leq 60$

Fluid properties are taken at film temperature: $T_f = (T_s + T_\infty)/2$

4.3.2 External Forced Convection

□ For Parallel flow over flat plates

Average convection coefficient for parallel flow over an **isothermal flat plate**:

Flow situation	Relationship $5 \times 10^5 \leq Re_L \leq 10^7$
Turbulent flow with negligible laminar component	$Nu = 0.037 Re_L^{0.8} Pr^{1/3}$ For $0.6 \leq Pr \leq 60$
Turbulent and laminar components are significant	$Nu = 0.037 (Re_L^{0.8} - 871) Pr^{1/3}$ For $0.6 \leq Pr \leq 60$

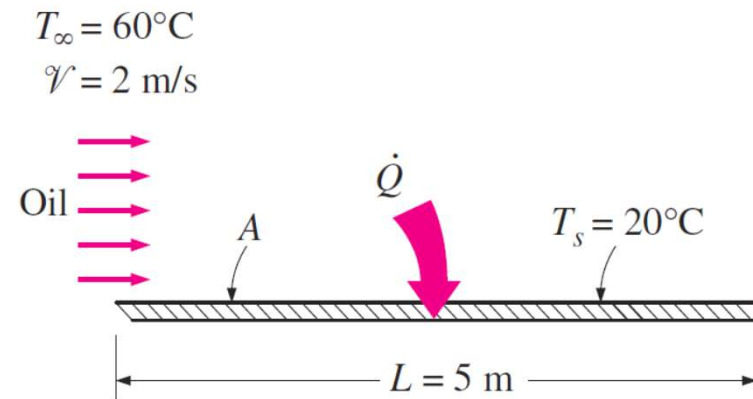
$$5 \times 10^5 = Re_{cr} = \frac{V x_{cr}}{\nu}$$

Fluid properties are taken at film temperature: $T_f = (T_s + T_\infty)/2$

4.3.2 External Forced Convection

EP# 3.3

Engine oil at 60°C flows over the upper surface of a 5-m-long flat plate whose temperature is 20°C with a velocity of 2 m/s. Determine the total drag force and the rate of heat transfer per unit width of the entire plate.



EP# 3.3 Solution

Assumptions

1. The flow is steady and incompressible.
2. The critical Reynolds number is $Re_{cr} = 5 \times 10^5$.

Properties

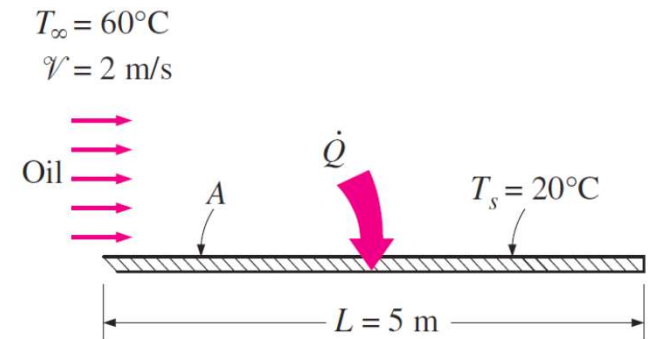
The properties of engine oil at the film temperature of $T_f = (T_s + T_\infty)/2 = (20 + 60)/2 = 40^\circ\text{C}$ are (Table A-13):

$$\rho = 876 \text{ kg/m}^3$$

$$Pr = 2962$$

$$k = 0.1444 \text{ W/m } ^\circ\text{C}$$

$$\nu = 2.485 \times 10^{-4} \text{ m}^2/\text{s}$$



4.3.2 External Forced Convection

EP# 3.4

To defrost ice accumulated on the outer surface of an automobile windshield, warm air is blown over the inner surface of the windshield. Consider an automobile windshield ($k_w=1.4$ W/m.K) with an overall height of 0.5 m and the thickness of 5 mm. The outside air ambient temperature is -20°C and the average air flow velocity over the outer shield surface is 80 km/h, while the ambient temperature inside the automobile is 25°C . Determine the convection heat transfer coefficient for the warm air blowing over the inner surface of the windshield, necessary to cause the accumulated ice to begin melting.

EP# 3.4 Solution

Assumptions

1. The flow is steady and incompressible.
2. The critical Reynolds number is $Re_{cr} = 5 \times 10^5$.

Properties

The properties of engine oil at the film temperature of $T_f = (T_s + T_\infty)/2 = (-20 + 0)/2 = -10^\circ\text{C}$ are (Table A–15):

$$\rho = 1.341 \text{ kg/m}^3$$

$$Pr = 0.3787$$

$$k = 0.02288 \text{ W/m }^\circ\text{C}$$

$$\nu = 1.252 \times 10^{-5} \text{ m}^2/\text{s}$$